

AMENDMENTS TO THE CLAIMS

Please amend claims 2-8 and cancel claims 1 and 13-16 as follows.

1. (Canceled)

2. (Currently Amended) ~~The~~A power tool, ~~as defined in claim 1 comprising:~~
 a tool bit;
 a tool body to which the tool bit is coupled;
 an actuating mechanism disposed in the tool body to drive the tool bit linearly by
means of pressure fluctuations so as to cause the tool bit to perform a predetermined operation,
wherein the actuating mechanism has a driving motor, a motion converting mechanism that
converts a rotating output of the driving motor to a linear motion, a piston linearly reciprocating
in a longitudinal direction of the tool bit via the motion converting mechanism, a striker
disposed in front of the piston to cause the tool bit a linear motion, a first chamber between the
striker and the piston, and a second chamber disposed adjacent to the piston within the tool body
in an opposite side of the first chamber; and
 a dynamic vibration reducer having a weight that reciprocates under a biasing force of an
elastic element to reduce vibration of the actuating mechanism, the weight being driven by
means of pressure fluctuations caused in the second chamber when the piston reciprocates,
 wherein the motion converting mechanism comprises a crank mechanism that drives the
striker by converting a rotating output of the driving motor to a linear motion in an axial
direction of the tool bit, and the second chamber is defined by a crank chamber that houses the
crank mechanism.

3. (Currently Amended) The power tool as defined in claim ~~1~~2, wherein, under loaded
driving conditions, in which a load associated with the predetermined power tool operation is
applied to the tool bit, the weight is allowed to be driven by means of fluctuating pressure
developed in the second chamber, while, under unloaded driving conditions, in which a load
associated with the predetermined power tool operation is not applied to the tool bit, the weight
is prevented from being driven by means of fluctuating pressure developed in the second
chamber.

4. (Currently Amended) The power tool as defined in claim 12, wherein, under loaded driving conditions, in which a load associated with the predetermined power tool operation is applied to the tool bit, the weight is allowed to be driven by means of fluctuating pressure developed in the second chamber, while, under unloaded driving conditions, in which a load associated with the predetermined power tool operation is not applied to the tool bit, the weight is prevented from being driven by means of fluctuating pressure developed in the second chamber and,

wherein the dynamic vibration reducer includes a first actuating chamber and a second actuating chamber that are defined on opposite sides of the weight within the body, and wherein, at least under the loaded driving conditions, the fluctuating pressure developed in the second chamber is introduced into the first actuating chamber, and the second actuating chamber can communicate with the outside.

5. (Currently Amended) The power tool as defined in claim 12, wherein, under loaded driving conditions, in which a load associated with the predetermined power tool operation is applied to the tool bit, the weight is allowed to be driven by means of fluctuating pressure developed in the second chamber, while, under unloaded driving conditions, in which a load associated with the predetermined power tool operation is not applied to the tool bit, the weight is prevented from being driven by means of fluctuating pressure developed in the second chamber and the fluctuating pressure developed in the second chamber is released to the outside of the power tool under the unloaded driving conditions by communicating the second chamber to the outside.

6. (Currently Amended) The power tool as defined in claim 12, wherein the tool bit comprises a hammer bit that performs a predetermined hammer operation by applying a linear impact force to a work piece.

7. (Currently Amended) The power tool as defined in claim 12, wherein the actuating mechanism includes a piston and a cylinder that slide relative to each other in an axial direction of the tool bit, wherein the tool bit reciprocates in its axial direction by the action of an air spring which is caused by relative movement of the piston and the cylinder, and wherein the weight is

disposed along a circumferential surface of the cylinder and can slide in the axial direction of the tool bit.

8. (Currently Amended) ~~The~~ A power tool as defined in claim 1, further comprising:

a tool bit;

a tool body to which the tool bit is coupled;

an actuating mechanism disposed in the tool body to drive the tool bit linearly by means of pressure fluctuations so as to cause the tool bit to perform a predetermined operation, wherein the actuating mechanism has a driving motor, a motion converting mechanism that converts a rotating output of the driving motor to a linear motion, a piston linearly reciprocating in a longitudinal direction of the tool bit via the motion converting mechanism, a striker disposed in front of the piston to cause the tool bit a linear motion, a first chamber between the striker and the piston, and a second chamber disposed adjacent to the piston within the tool body in an opposite side of the first chamber;

a dynamic vibration reducer having a weight that reciprocates under a biasing force of an elastic element to reduce vibration of the actuating mechanism, the weight being driven by means of pressure fluctuations caused in the second chamber when the piston reciprocates; and

a cylinder that houses the striker such that the striker slidably reciprocates within the cylinder,

wherein the cylinder moves between a first position near the tool holder and a second position remote from tool holder than the first position, and under loaded driving conditions in which a load associated with the predetermined operation is applied to the tool bit, the cylinder moves to the second position so as to allow the weight to be driven by means of fluctuating pressure within the second chamber, while, under unloaded driving conditions in which a load associated with the predetermined operation is not applied to the tool bit, the cylinder moves to the first position so as to prevent the weight from being driven by means of fluctuating pressure within the second chamber.

9. (Previously Presented) The power tool as defined in claim 8, wherein under the loaded driving conditions, the cylinder moves to the second position so as to allow the striker to be driven by the action of the air spring function of the first chamber, while, under unloaded driving

conditions, the cylinder moves to the first position, so as to prevent the striker from being driven by the action of the air spring function of the first chamber.

10. (Previously Presented) The power tool as defined in claim 9, wherein under the loaded driving conditions, the weight is allowed to be driven by fluctuating pressure within the second chamber after the striker is allowed to be driven by the action of the air spring function of the first chamber.

11. (Previously Presented) The power tool as defined in claim 8, further comprising an air vent that can communicate the second chamber with the outside, wherein when the cylinder moves to the second position, the air vent is closed so as to allow the weight to be driven, and when the cylinder moves to the first position, the air vent is opened so as to prevent the weight to be driven.

12. (Previously Presented) The power tool as defined in claim 8, further comprising an air vent that can communicate the first chamber with the outside, wherein the air vent is closed when the cylinder moves to the second position and the air vent is opened when the cylinder moves to the first position.

13-16. (Canceled)